

Room Impulse Response Interpolation from a Sparse Set of Measurements Using a Modal Architecture

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Motivation

- Dynamic reverberation with changing source and listener position is a requirement for realistic room acoustics simulation.
 - VR
Information about geometry, materials is available.
 - AR
Missing geometries, inaccurate materials.
Sparse representation of the room can be obtained.

Objectives

- Generate a continuous RIR space utilizing a sparse dataset of RIRs from a room.
- Smooth, real-time interpolation, extrapolation and morphing between measured positions.
- Accurate spatial interpolation of perceptually relevant low frequency modes in rooms with simple geometries having non-rigid walls.

Related Work

- RIR interpolation with Dynamic Time Warping.¹
 - Not a function of source/listener position.
- Compressed Sensing²
 - Reconstructs early part of RIR only.
 - Requires accurate calibration of microphone array.
- Common acoustical-pole and residue model³ (derived from room modes).
 - Parameterized by source and listener locations.
 - Physics is well known for rectangular rooms.

[1] Masterson et al. "Acoustic impulse response interpolation for multichannel systems using dynamic time warping.", *AES 2009*

[2] Mignot et al. "Room reverberation reconstruction: Interpolation of the early part using compressed sensing." *IEEE TALSP 2013*.

[3] Haneda et al. "Common-acoustical-pole and residue model and its application to spatial interpolation and extrapolation of a room transfer function." *IEEE TASLP, 99*

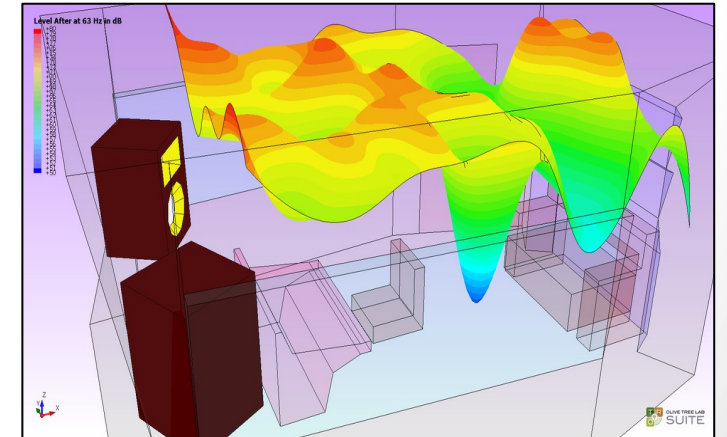
Room Modes I

- Standing waves are solutions to the 3D wave equation
- RIR can be characterized as a sum of M room modes⁴

$$h(x, y, z, t) = \sum_{m=1}^M \gamma_m(x, y, z) e^{(j\omega_m - \alpha_m)t}$$

Complex mode amplitudes depend on spatial location of source/listener

Mode frequencies and dampings determine the time response



- RIRs can be efficiently synthesized with M parallel biquads

$$H(z) = \sum_{m=1}^M \frac{\text{Re}(\gamma_m) - e^{-\alpha_m} \text{Re}(\gamma_m e^{-j\omega_m}) z^{-1}}{1 - 2e^{-\alpha_m} \cos(\omega_m) z^{-1} + e^{-2\alpha_m} z^{-2}}$$

[4] Abel et al. , "A modal architecture for reverberation with application to room acoustics modeling", AES 2014.

Room Modes II

- Complex mode amplitudes are the solution to Helmholtz equation

$$\nabla^2 p + k^2 p = 0; \text{Wave number } k = \frac{\omega m}{c}$$

- For rigid walls

$$p(x, y, z) = C \cos(k_x x) \cos(k_y y) \cos(k_z z)$$

From boundary conditions $k_\mu = \frac{n_\mu \pi}{L_\mu}$

$$k = \sqrt{k_x^2 + k_y^2 + k_z^2}$$

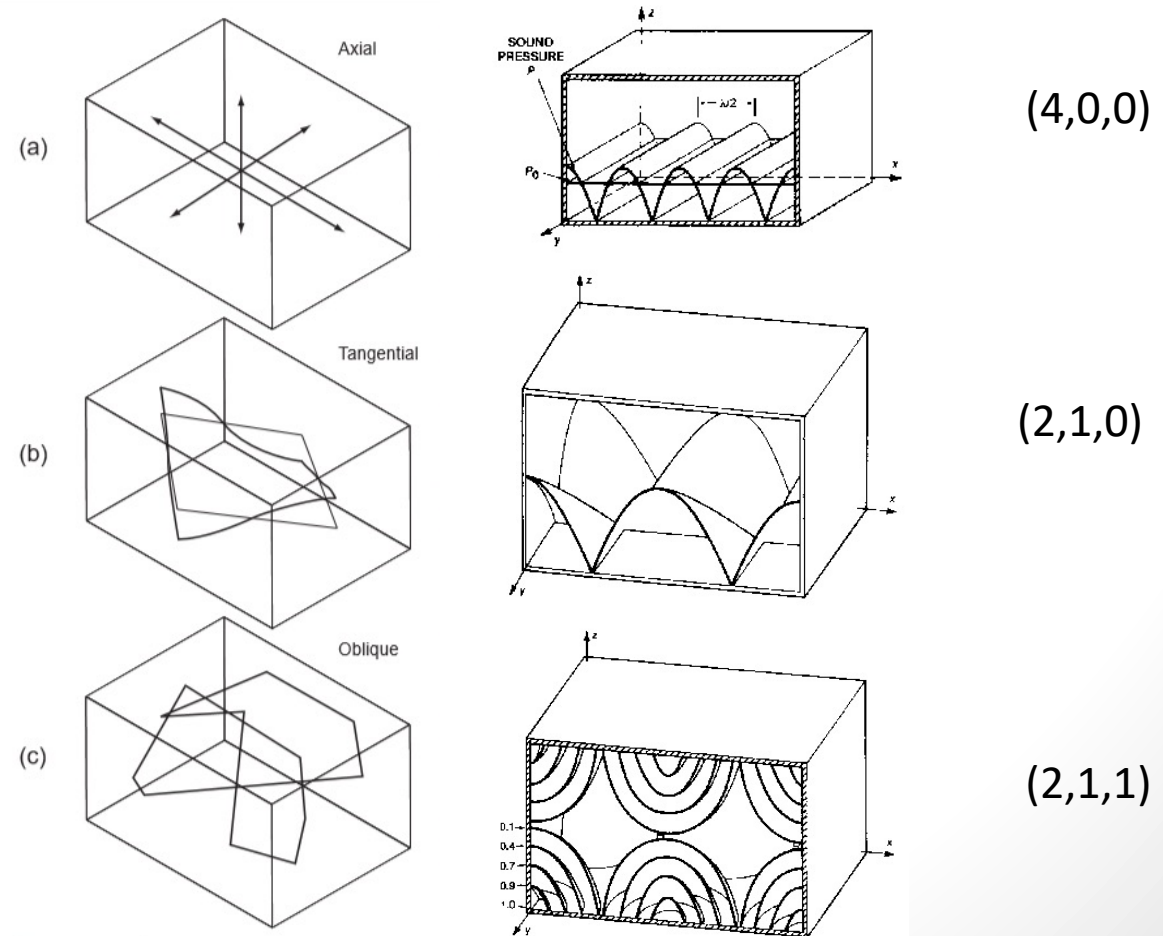
- For non-rigid walls with finite impedance

$$p(\mu) = C_\mu \exp(ik_\mu \mu) + D_\mu \exp(-ik_\mu \mu)$$

$$k_\mu = \frac{n_\mu \pi}{L_\mu} + j\delta_u \quad \text{Wall absorption}$$

$$p(x, y, z) = p(x)p(y)p(z)$$

Pressure distributions

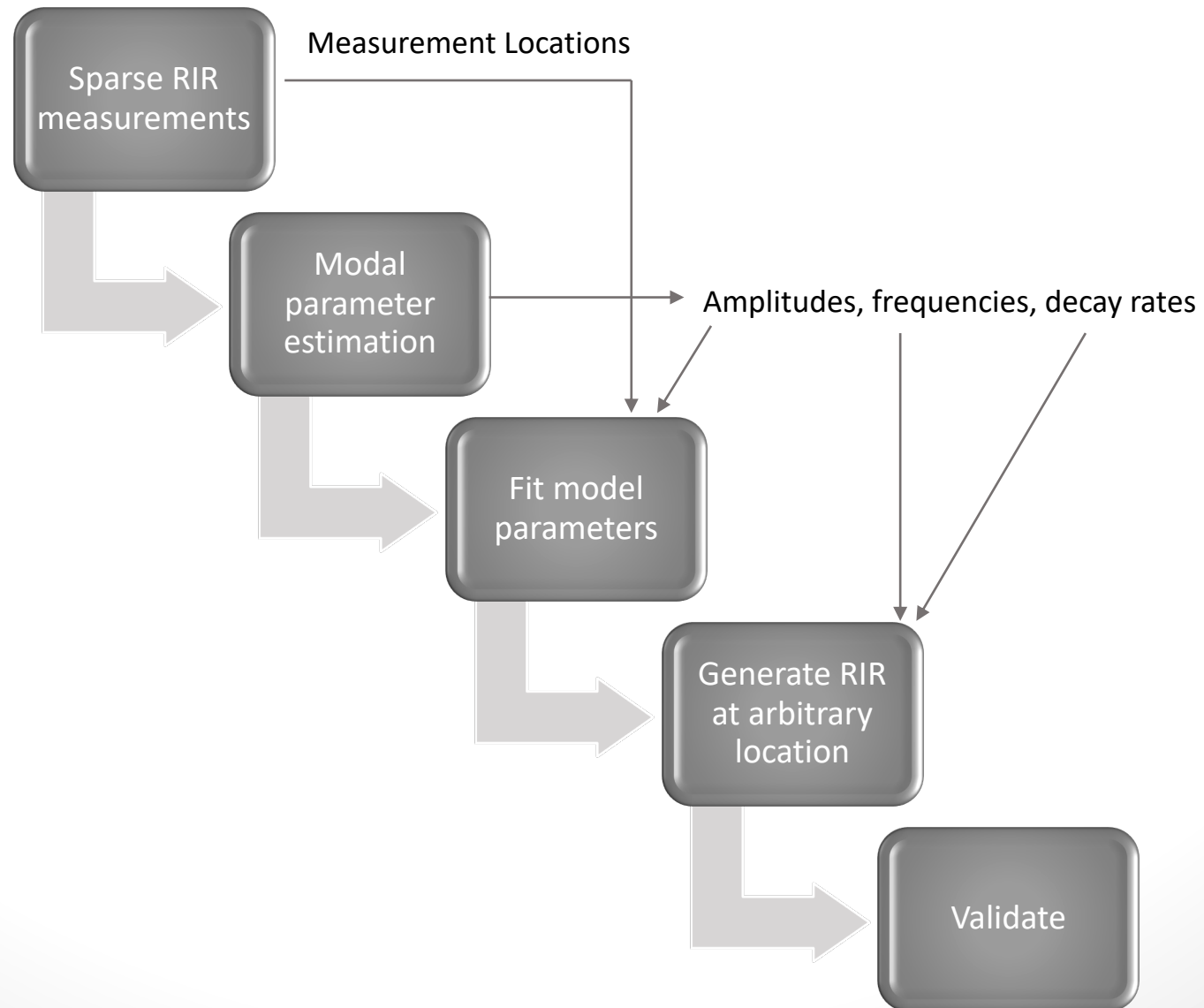


Kuttruff, Heinrich. *Room acoustics*, 4th Edition, Spon Press (2000).

Room Modes III

- Analytic solution for pressure distribution as a function of spatial location in a room (with non-rigid walls) requires
 - knowledge of room dimensions.
 - wall absorption, which is frequency dependent.
- Assuming we don't have this information, can we fit parameters to the measurements?

Pipeline



Modal Estimation

For a grid of RIR measurements

- Average RIRs over all locations and estimate common mode frequencies ω_m and decay rates α_m using subband ESPRIT⁵
- Calculate unique set of M mode amplitudes for L locations in grid using linear least squares.

$$\begin{pmatrix} h_1(0) & h_2(0) & \cdots & h_L(0) \\ h_1(1) & h_2(1) & \cdots & h_L(1) \\ \vdots & \vdots & \vdots & \vdots \\ h_1(T) & h_2(T) & \cdots & h_L(T) \end{pmatrix} = \begin{pmatrix} 1 & \cdots & 1 \\ e^{-(j\omega_1 - \alpha_1)1} & \cdots & e^{-(j\omega_M - \alpha_M)1} \\ \vdots & \ddots & \vdots \\ e^{-(j\omega_1 - \alpha_1)T} & \cdots & e^{-(j\omega_M - \alpha_M)T} \end{pmatrix} \begin{pmatrix} \gamma_{11} & \cdots & \gamma_{L1} \\ \gamma_{12} & \cdots & \gamma_{L2} \\ \vdots & \vdots & \vdots \\ \gamma_{M1} & \cdots & \gamma_{ML} \end{pmatrix}$$

[5] Kereliuk et al. "Modal analysis of room impulse responses using subband ESPRIT." In *Proceedings of the International Conference on Digital Audio Effects*. 2018.

Optimization - Model

- Goal - Given a sparse set of impulse responses on a 2D plane, find the complex mode amplitudes and parameterize them as functions of (x,y) location

- General solution to 2D Helmholtz equation for non-rigid walls is

$$\widehat{\gamma}_m(x, y) = C_{1m} e^{-j(k_{xm}x + k_{ym}y)} + D_{1m} e^{j(k_{xm}x + k_{ym}y)} + C_{2m} e^{-j(k_{xm}x - k_{ym}y)} + D_{2m} e^{j(k_{xm}x - k_{ym}y)}$$

- For each location in grid $(x_1, y_1), \dots, (x_L, y_L)$

$$\begin{pmatrix} \widehat{\gamma}_m(x_1, y_1) \\ \vdots \\ \widehat{\gamma}_m(x_L, y_L) \end{pmatrix} = \begin{pmatrix} e^{-j(k_{xm}x_1 + k_{ym}y_1)} & e^{j(k_{xm}x_1 + k_{ym}y_1)} & e^{-j(k_{xm}x_1 - k_{ym}y_1)} & e^{-j(k_{xm}x_1 - k_{ym}y_1)} \\ \vdots & \vdots & \vdots & \vdots \\ e^{-j(k_{xm}x_L + k_{ym}y_L)} & e^{j(k_{xm}x_L + k_{ym}y_L)} & e^{-j(k_{xm}x_L - k_{ym}y_L)} & e^{-j(k_{xm}x_L - k_{ym}y_L)} \end{pmatrix} \begin{pmatrix} C_{1m} \\ D_{1m} \\ C_{2m} \\ D_{2m} \end{pmatrix}$$

$$\widehat{\gamma}_m = \underbrace{U_m(k_{xm}, k_{ym})}_{\text{Non-linear in wave numbers}} \underbrace{C_m}_{\text{Linear in constants}}$$

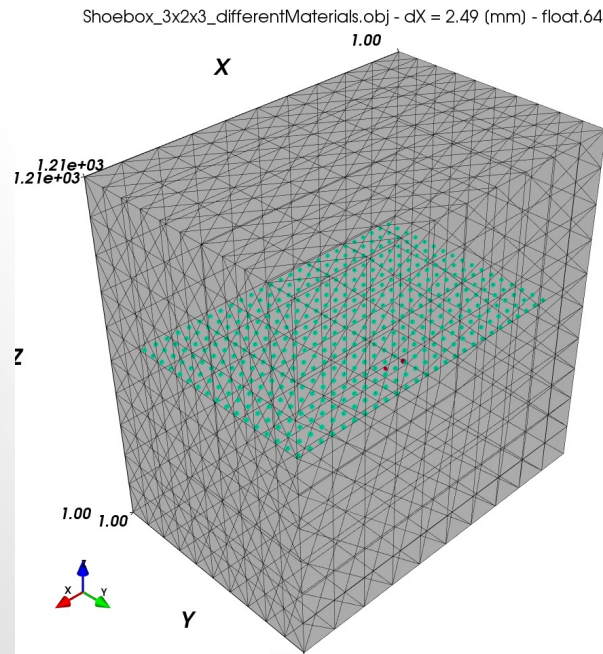
Non-linear in wave numbers

Optimization – Sequential update

- Find wave numbers and constants for each mode from the measured set of mode amplitudes.
- For modes, $m = 1, 2, \dots, M_c$
 - Initialize : $k_{x_m}(0), k_{y_m}(0) = \frac{\omega_m + j\alpha_m}{\sqrt{2}c}$
 - Repeat, till convergence
 - Update constants : $c_m(i) = U_m(i-1)^+ \gamma_m$
 - Update wave numbers : $\operatorname{argmin}_{k_{x_m}, k_{y_m}} \left| \left| 20 \log_{10}(\gamma_m) - 20 \log_{10}(U_m(i)c_m(i)) \right| \right|^2$
 - Bounds : $0 \leq k_{x_m}, k_{y_m} \leq \frac{\omega_m + j\alpha_m}{c}$
- Calculate mode amplitudes at arbitrary 2D location in room
 - $\widehat{\gamma}_m(x, y) = U_m(k_{x_m}^*, k_{y_m}^*) c_m^*$

Validation

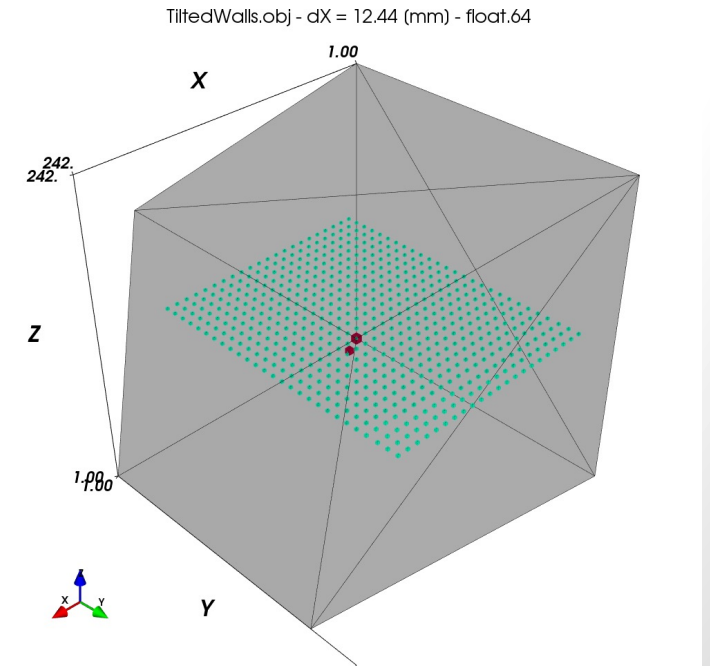
- Accurate simulation of small room acoustics with FDTD simulator.⁶
- Omni source placed at (0.3,0.2,0.1). Receivers placed in XY plane at resolution of $d = 20\text{cm}$ at a nominal standing height of 1.7m.
- Maximum mode frequency corresponding to M_c is $\frac{c}{2d} = 866\text{ Hz}$.



Left – Shoebbox room of dimensions $3 \times 2 \times 3\text{ m}^3$

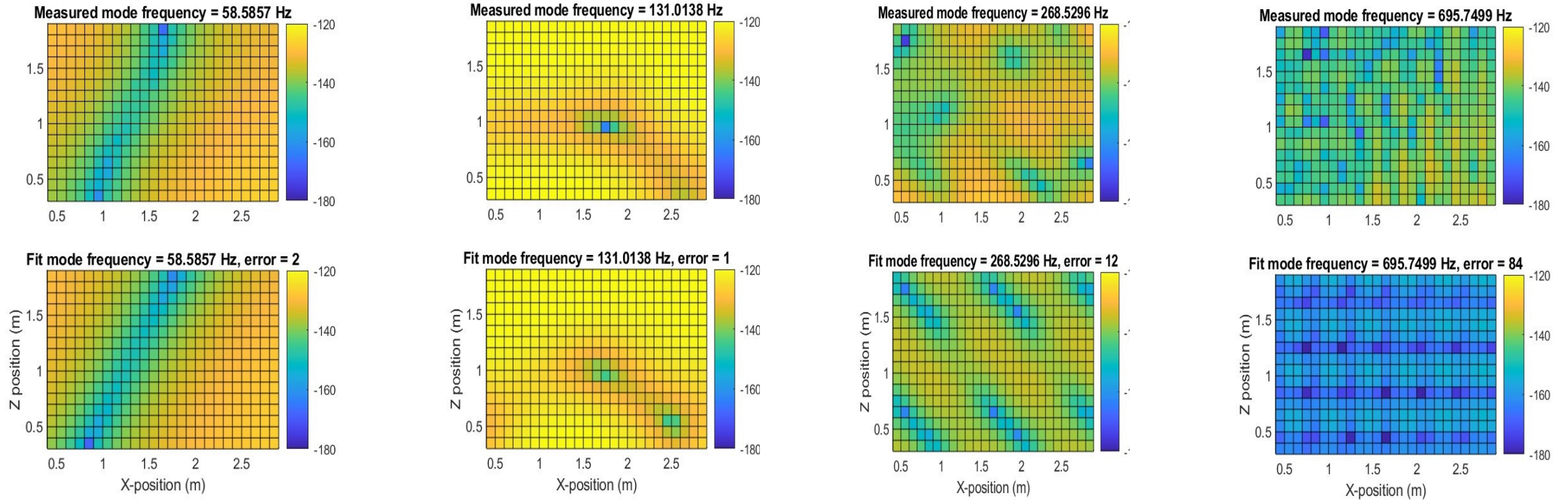
Right – Room with no parallel walls, 3m on longest edge

Front and back wall admittance - 0.9
Left and right wall admittance - 0.8
Floor and ceiling admittance - 0.99



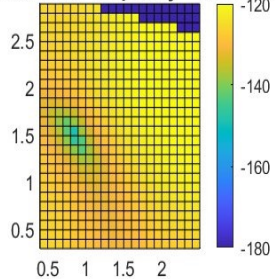
[6] Saarelma, J., Califa, J. and Mehra, R. "Challenges of distributed real-time finite-difference time-domain room acoustic simulation for auralization." In *2018 AES International Conference on Spatial Reproduction-Aesthetics and Science*. Audio Engineering Society.

Modal fit for shoebox room – 442 mics

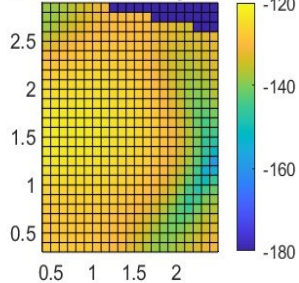


Modal fit for room with tilted walls – 594 mics

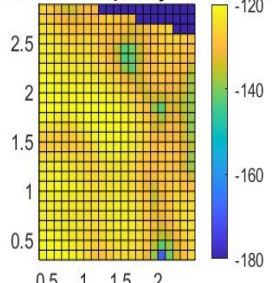
Measured mode frequency = 62.1668 Hz



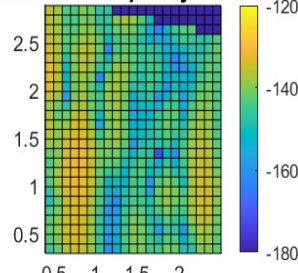
Measured mode frequency = 125.8605 Hz



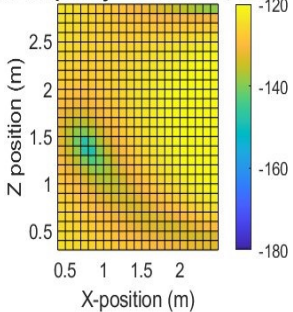
Measured mode frequency = 326.3342 Hz



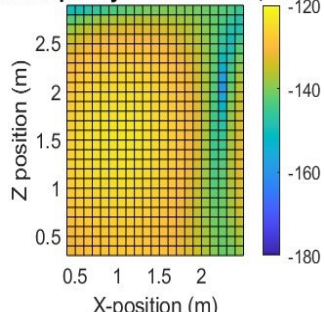
Measured mode frequency = 482.1765 Hz



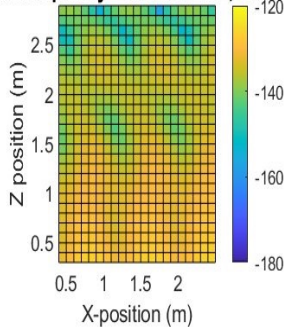
Fit mode frequency = 62.1668 Hz, error = 223



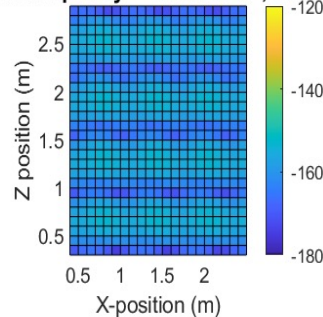
Fit mode frequency = 125.8605 Hz, error = 18



Fit mode frequency = 326.3342 Hz, error =



Fit mode frequency = 482.1765 Hz, error = 193



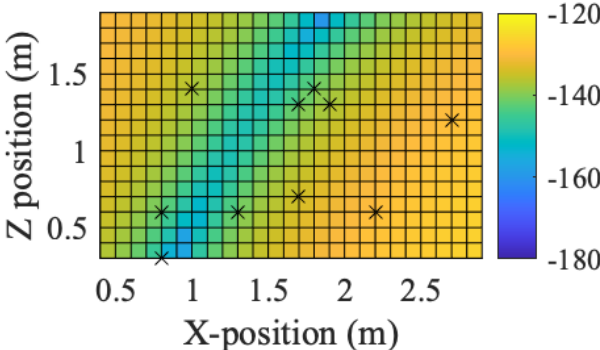
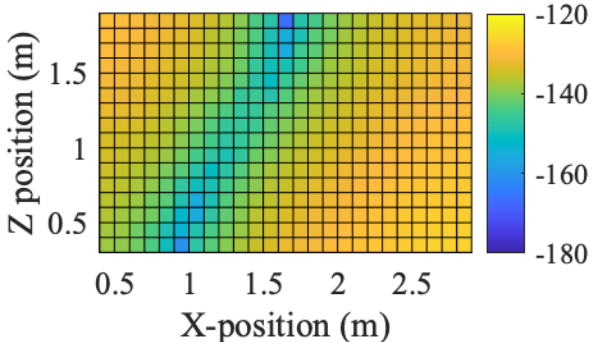
Experiment with Sparse Microphones

- To replicate sparsity, vary number of microphones in each simulated room from **5 to 50**.
- Run $N_{tr} = 100$ trials for each set, placing the microphones in different configurations.
- Calculate and plot
 - **Mean Structural Similarity Index (MSSIM)** – metric between [0,1] to measure similarity between actual and fit mode shapes.
 - **Absolute Mean Spectral Difference Error (AMSDE)** -

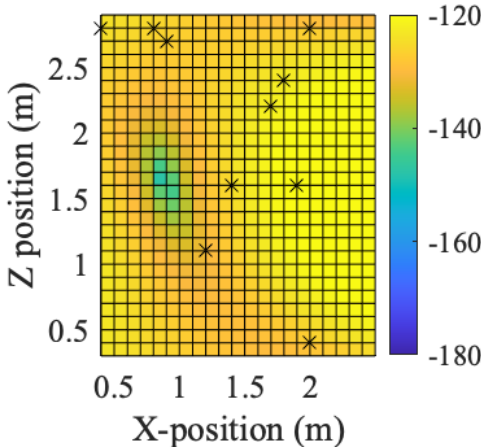
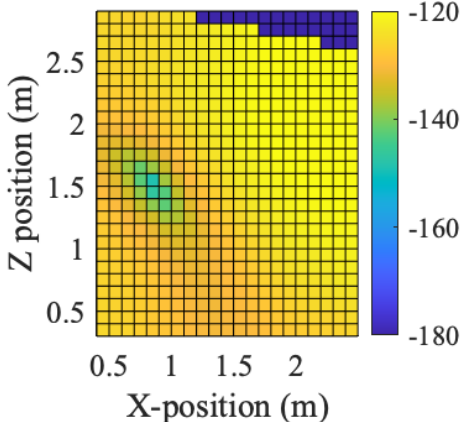
$$\frac{1}{N_{tr}} \sum_{n=1}^{N_{tr}} \left| 20 \log_{10} \frac{\sum_{l=1}^L H_{l,n}(\omega)}{\sum_{l=1}^L \hat{H}_{l,n}(\omega)} \right|$$

Results

Modal fits with 10 microphones

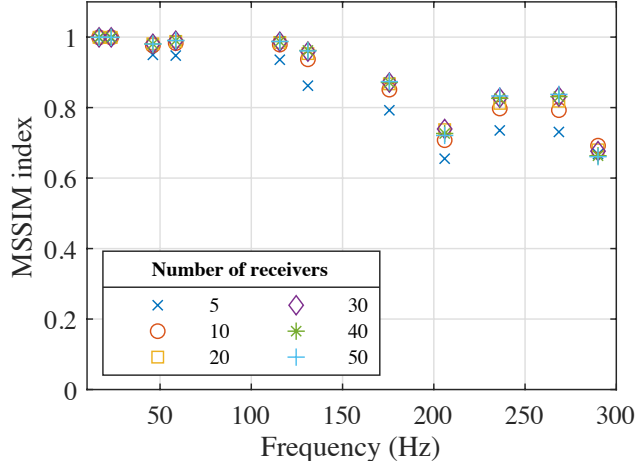


Shoobox, Mode at 58 Hz

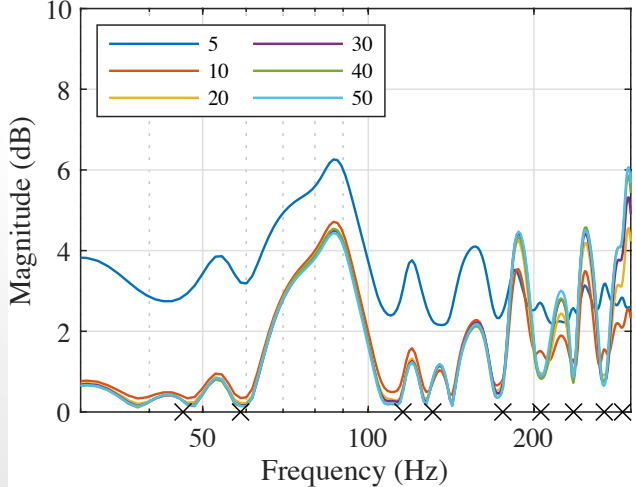
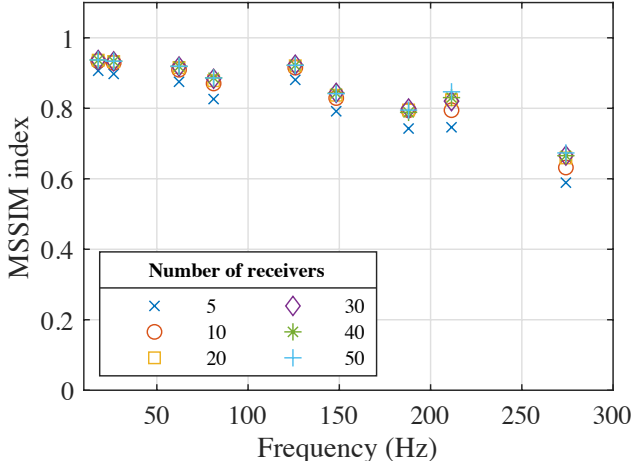


Tilted Walls, Mode at 62 Hz

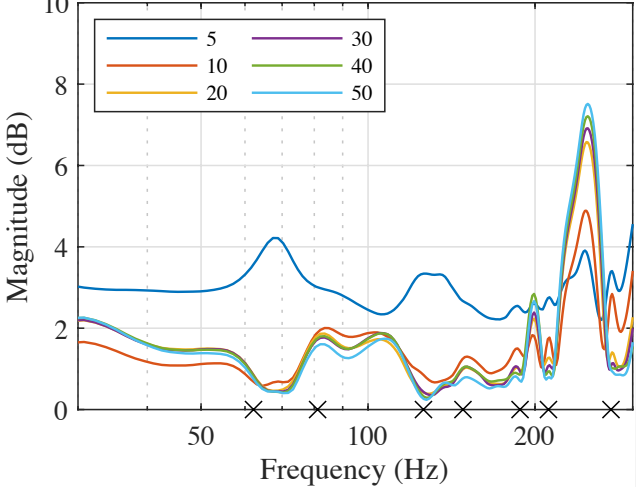
Results



MSSIM



AMSDE



Shoebox

Tilted Walls

Conclusion

- Proposed method is capable of accurate RIR interpolation in the lower frequencies with a very small number of randomly distributed microphones.
- Only a handful of parameters need to be stored for characterizing the low-frequency spatial response of a room - mode frequencies, dampings, two wave numbers and four constants for each mode.
- RIR can be interpolated very efficiently in real-time once these parameters have been estimated offline.

Future Work

- Comparison with State-of-the-art DL method.⁷
- Perceptual evaluation.
- Extend model beyond low frequencies.

[7] Lluís, F., Martínez-Nuevo, P., Bo Møller, M. and Ewan Shepstone, S. "Sound field reconstruction in rooms: Inpainting meets super-resolution." *The Journal of the Acoustical Society of America*, 148(2), pp.649-659, 2020.



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